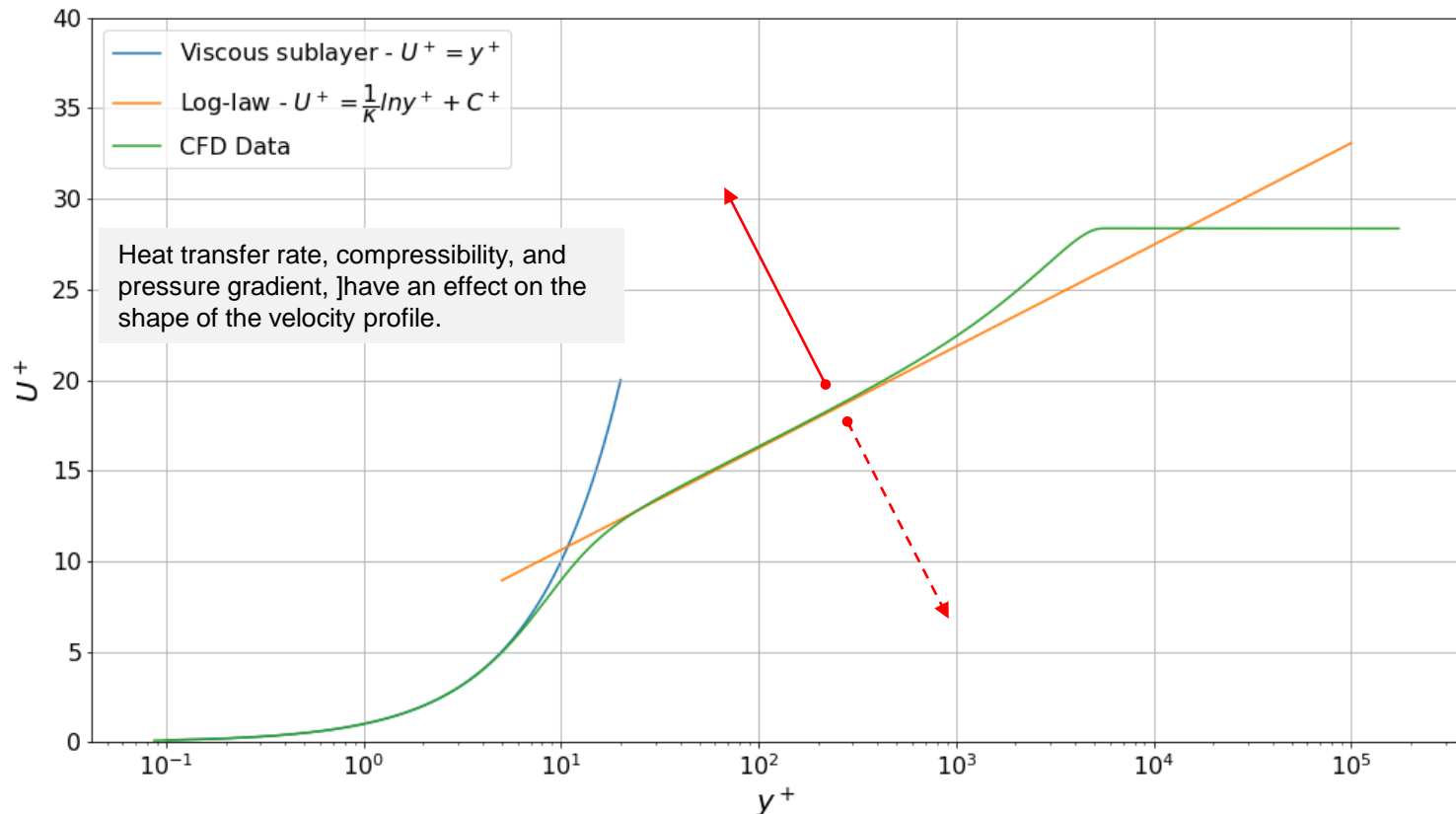


Wall functions – Additional observations

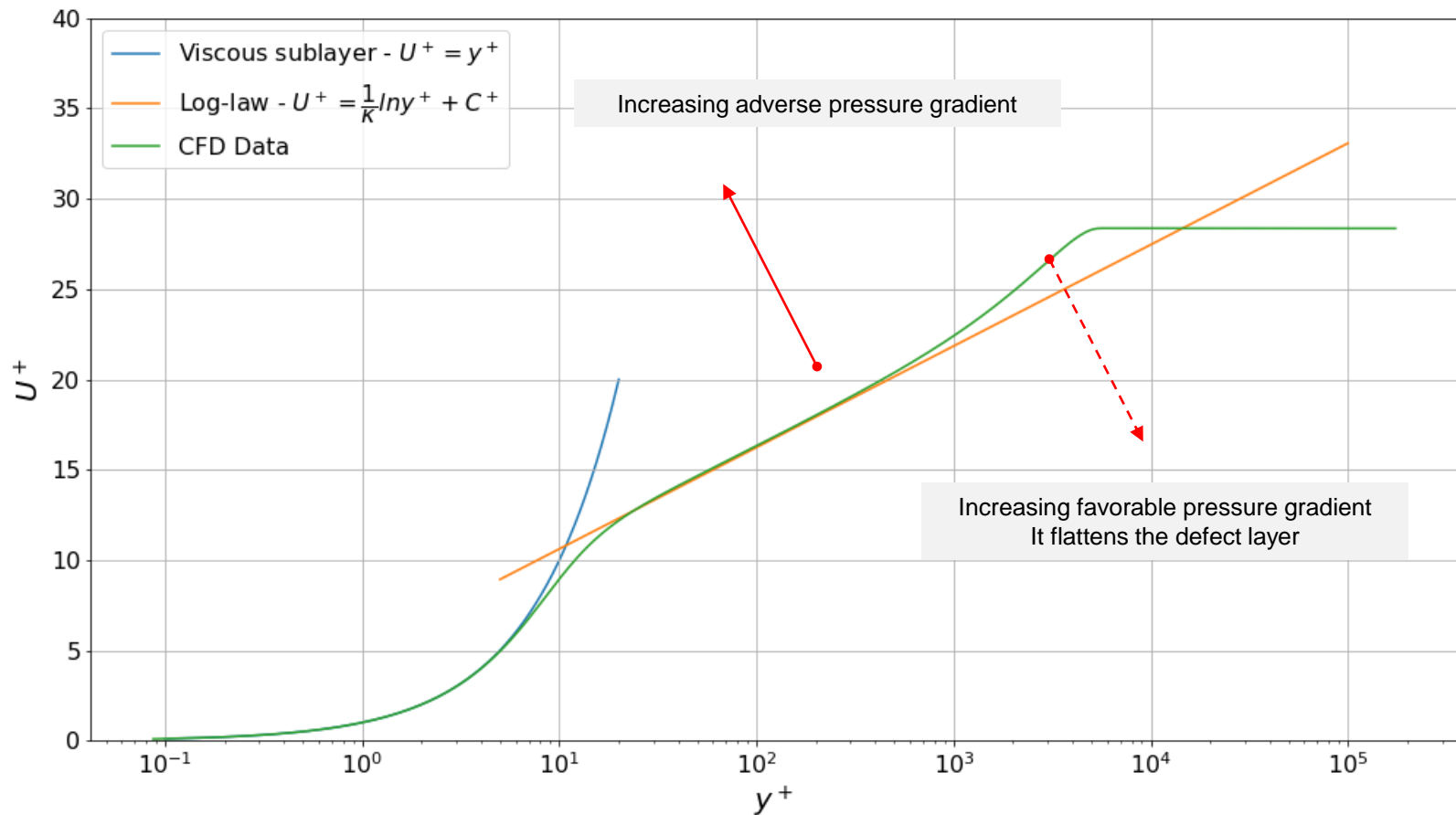
Wall functions – Additional observations

- When studying the boundary layer, heat transfer rate, compressibility, pressure gradient, mach number, and surface roughness (among many factors), they all have an effect on the velocity profile.
- It is not obvious what effect each of these parameters have on the velocity profile.
- Hereafter, we present a few observations or general guidelines (use with care).



Wall functions – Additional observations

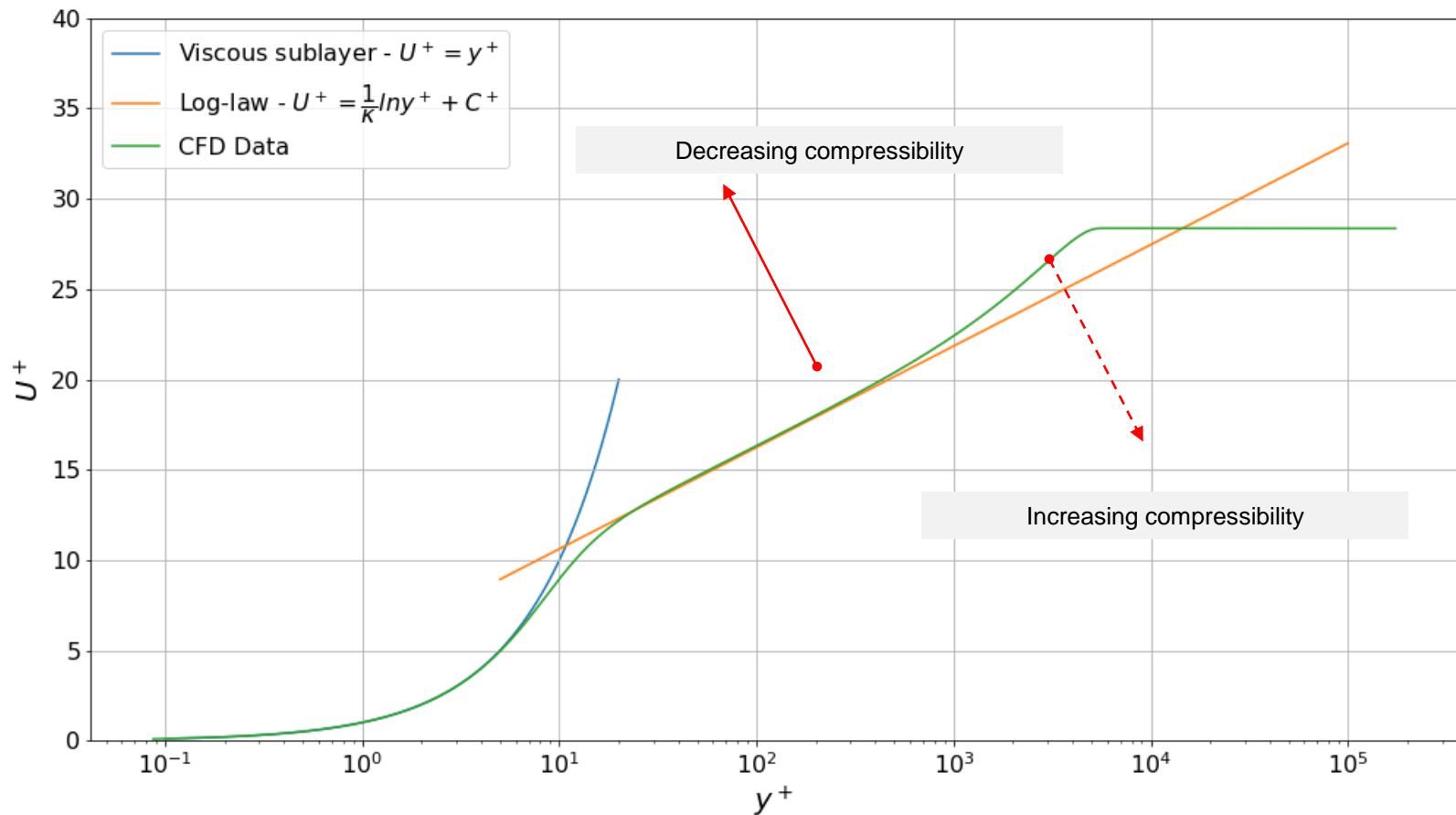
- Effect of adverse pressure gradient on the law of the wall.



- The boundary layer thickens, and the skin friction decreases as the adverse pressure gradient is increased.

Wall functions – Additional observations

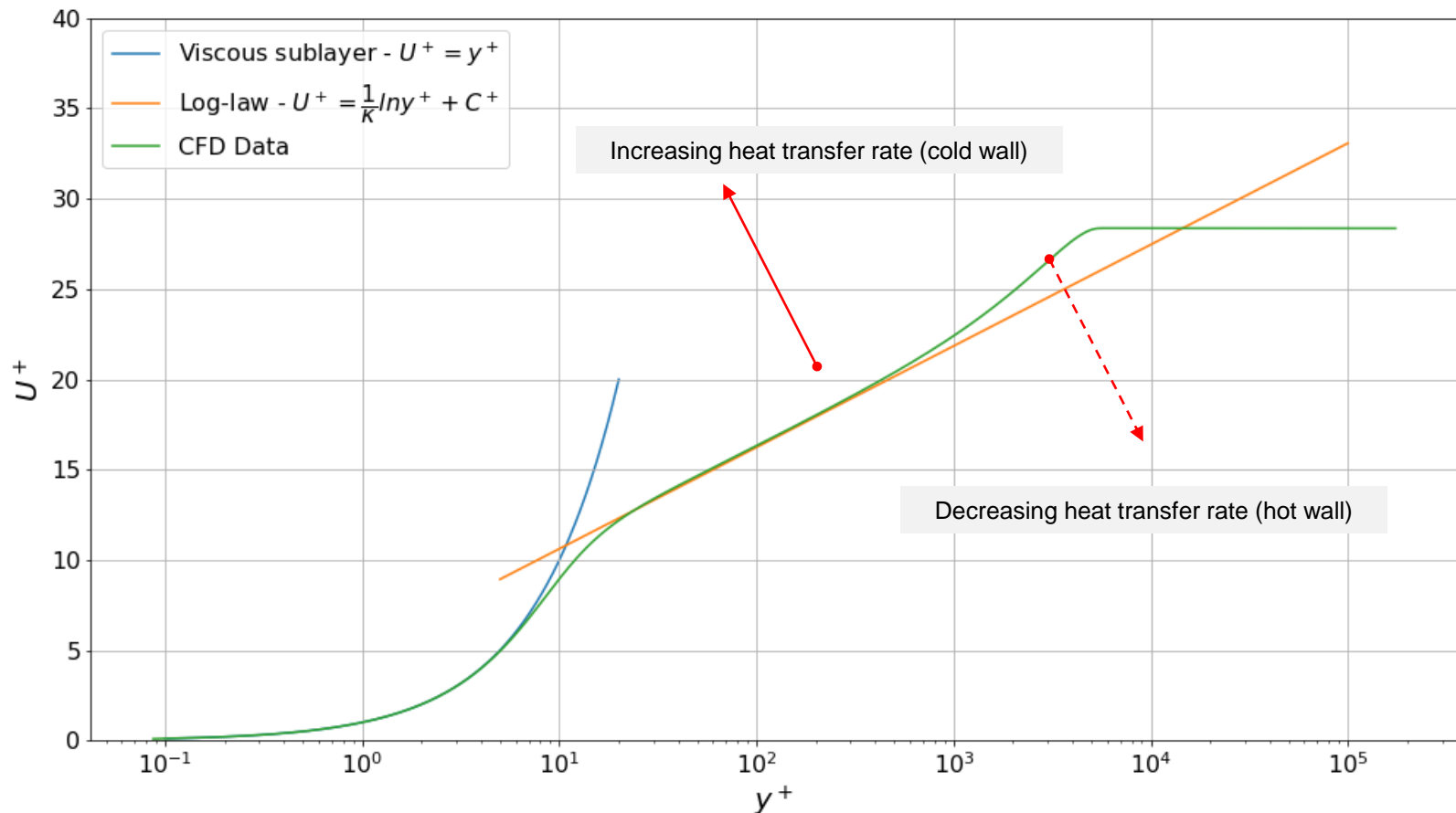
- Effect of compressibility on the law of the wall.



- Increasing compressibility causes the skin friction to decrease.

Wall functions – Additional observations

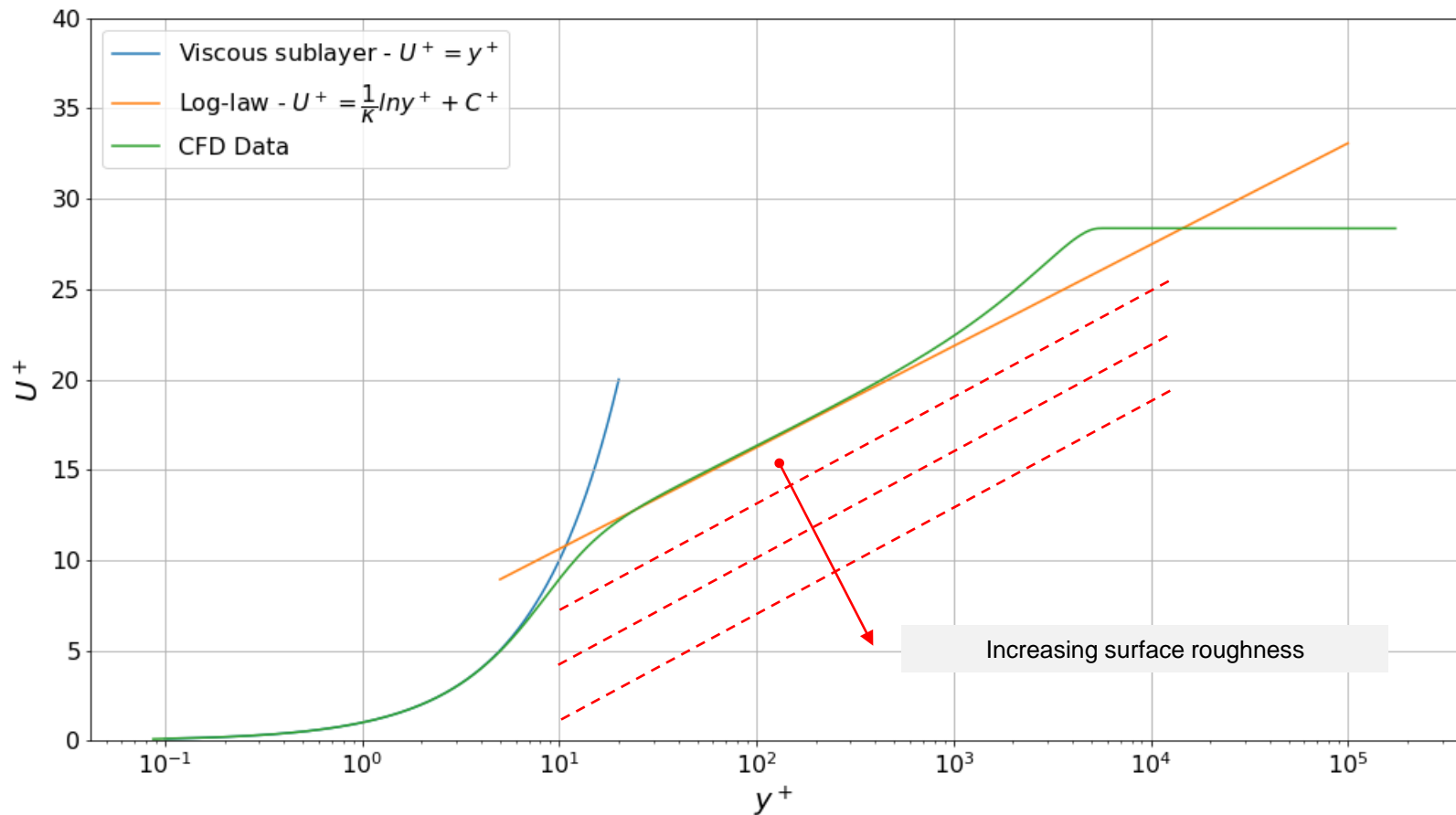
- Effect of heat transfer on the law of the wall.



- Notice that depending on the Prandtl number, the boundary layer can be thicker or thinner.
- A cold wall creates a thinner boundary layer and increases the skin friction
- A hot wall thickens the boundary layer and decreases the skin friction.

Wall functions – Additional observations

- Effect of surface roughness on the law of the wall.



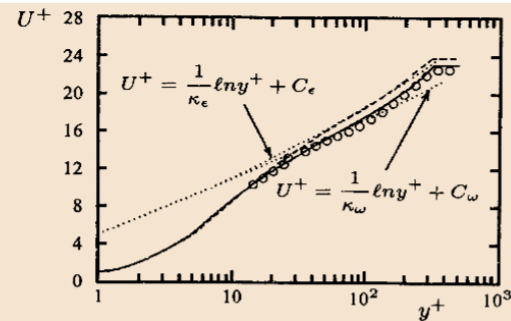
- Surface roughness increases the skin friction.
- It shifts the velocity profile downwards.

Wall functions – Additional observations

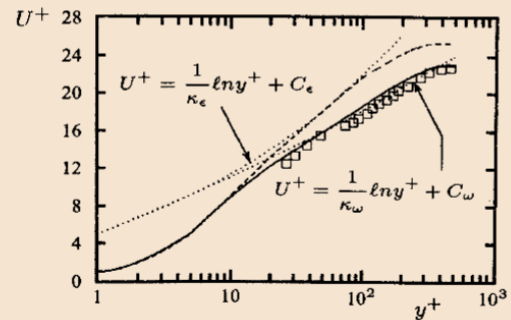
- When dealing with high-speed flows and heat transfer, the coefficients appearing in the law of the wall relations and closure relations, have a strong dependence on the Mach number and the heat transfer rate.
- No need to mention that the physical properties also depend on the Mach number and heat transfer rate.

$$u^* = \begin{cases} y^* & y^* < 11.225 \\ \frac{1}{\kappa} \ln(y^*) + C & y^* > 11.225 \end{cases}$$

$$T^* = \begin{cases} Pr y^* & y^* < 5 \\ Pr_t \left[\frac{1}{\kappa} \ln(Ey^*) + P \right] & y^* > 30 \end{cases}$$



(a) Mach 4.5



(b) Mach 10.3

Figure 5.3: Computed and measured velocity profiles for compressible flat-plate boundary layers: — Wilcox (2006) k - ω ; - - Chien k - ϵ ; \circ Coles; \square Watson.

Wall functions – Additional observations

- In the case of pressure gradients and flow separation (non-equilibrium conditions), the law of the wall can be sensitized to pressure gradients effects [1].

$$\frac{\tilde{U} C_\mu^{1/4} k^{1/2}}{\tau_w / \rho} = \frac{1}{\kappa} \ln \left(E \frac{\rho C_\mu^{1/4} k^{1/2} y}{\mu} \right)$$

- Where,

$$\tilde{U} = U - \frac{1}{2} \frac{dp}{dx} \left[\frac{y_v}{\rho \kappa k^{1/2}} \ln \left(\frac{y}{y_v} \right) + \frac{y - y_v}{\rho \kappa k^{1/2}} + \frac{y_v^2}{\mu} \right]$$

$$y_v = \frac{\mu y_v^*}{\rho C_\mu^{1/4} k_p^{1/2}} \quad y_v^* = 11.225$$

- This correction is recommended for use in complex flows involving separation, reattachment, and impingement where the mean flow and turbulence are subjected to pressure gradients and rapid changes.
- In such flows, improvements can be obtained, particularly in the prediction of wall shear (skin-friction coefficient) and heat transfer (Nusselt or Stanton number).

Wall functions – Additional observations

- The non-dimensional temperature T^* , can be further improved by adding the viscous heating contribution [1],

$$T^* = T_c^* + \frac{D}{q}$$

- Where,

$$T_c^* = \begin{cases} Pr y^* & y^* < y_l^* \\ Pr_T \left[\frac{1}{\kappa} \ln(Ey^*) + P \right] & y^* > y_l^* \end{cases}$$

$$D = \begin{cases} \rho u^* \frac{1}{2} Pr U_p^2 & y^* < y_l^* \\ \rho u^* \frac{1}{2} [Pr_t U_p^2 + (Pr - Pr_t) U_c^2] & y^* > y_l^* \end{cases}$$

- Where U_c is the mean velocity at the intersection point y_l^* .
- This correction is recommended for highly compressible flows, where heating by viscous dissipation can have a strong influence in the temperature distribution in the near-wall region.